

Postbuckling of Marine Stiffened Composite Plates with Initial Geometric Imperfections Using Progressive Failure Analysis

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Abstract

This work explores the postbuckling behavior of a marine stiffened composite plate in the presence of initial imperfections. The imperfection shapes are derived from buckling mode shapes and their combinations. Thereafter, these imperfection shapes are applied to the model, and nonlinear large deflection finite element and progressive failure analyses are performed in ANSYS 18.2 software. The Hashin failure criterion is employed to model the progressive failure in the stiffened composite plate. The effect of the initial geometric imperfection on the stiffened composite plate is investigated by considering various imperfection patterns and magnitudes. Results show that when the magnitude of the imperfection is 20 mm, the ultimate strength of the stiffened composite plate decreases by 31%. Moreover, global imperfection shapes are found to be fundamental in determining the ultimate strength of stiffened composite plates and their postbuckling.

Keywords Initial geometric imperfection · Laminated composite plate · Postbuckling behavior · Nonlinear finite element method · Progressive damage method · Hashin damage criteria

1 Introduction

Using stiffened laminated composite plates in marine structures can decrease their weight, increase their production speed, and decrease their life cycle cost. Thus, the application of laminated composite materials in offshore and ship structure fields has broadened rapidly. Many marine structures are made of composite materials. In ships, composite materials are used to manufacture hull structures, decks, bulkheads, and topsides. In general, these structures consist

of plates that are stiffened in two orthogonal directions. During a ship's lifetime, its stiffened plates are exposed to various loads, with hull girder stress being the dominant one. Such type of load is due to the bending of the whole ship hull, similar to a simple beam.

As a result of the precise and advanced construction methods utilized in the last decade, relatively thin and light stiffened composite plates have been adopted in ship structures. Therefore, these structures have become increasingly prone to the buckling behavior of ship decks or floor structures under in-plane hull girder stress. Zhu et al. (2015) experimentally studied the effect of stiffeners' stiffness and plate thickness on the ultimate strength of marine composite stiffened panels and concluded that plate thickness exerts a considerable effect on the ultimate strength of composite stiffened panels. To comprehend the postbuckling behavior of composite stiffened panels, Bai et al. (2018) investigated the strength of a composite stiffened panel by conducting some experiments.

Barsotti et al. (2020) indicated that the macro-mechanical aspects of mechanical properties are widely used in the design of composite structures because of their simplicity. Progressive damage methodologies are based on macro-mechanical properties and are commonly utilized to investigate the strength of laminated composite plates and

Article Highlights

- The effect of initial geometric imperfection mode shapes and magnitude on the ultimate strength of the stiffened composite panels are studied.
- The 11th first buckling mode shapes and a series of combinations of them are assumed as initial geometric imperfections.
- The postbuckling and damage are modeled using the progressive damage method and nonlinear finite element method.

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stiffened plates. In their review, Murugesan and Rajamohan (2017) stated that because of the failure behavior of layered composite structures, progressive failure methods are highly appropriate. On the basis of a progressive failure method for analyzing laminated composites, Chen and Guedes Soares (2007a, b) developed a technique for investigating the ultimate strength of stiffened composite panels and derived a nonlinear formulation for laminated shell finite elements. Huang et al. (2015) generated a 3D finite element curved beam model to analyze grid stiffened curved panels by using a progressive failure method. The degree of freedom in this beam model is compatible with that of shell elements, hence, the number of unknowns decreases.

Pal and Bhattacharyya (2007), Pal and Ray (2002), and Namdar and Darendeliler (2017) used progressive failure analysis to evaluate the strength of laminated composite plates under transverse loading and shear loading. Priyadarshani et al. (2017) investigated the ultimate strength of various composite stiffened panels with cutouts by using a finite element method (FEM) and performing experiments. Kong et al. (1998) conducted a series of compression tests on a stiffened composite plate, performed nonlinear finite element analysis, and implemented progressive failure methods to model the compression test results.

The uncertainties correlated with laminated composite structures exert a significant impact on their postbuckling strength. These uncertainties could be presented in material properties, ply thickness, fiber orientation, geometric imperfection, etc. No systematic guideline or standard is available to describe how to consider these uncertainties, especially in the field of marine composite structures. Thus, the effects of various uncertainties on the ultimate strength of stiffened laminated composite plates need to be studied. Considering these uncertainties, Chen and Guedes Soares (2007a, b; 2008a, b), respectively, presented a reliability finite element method and a stochastic finite element method to assess the reliability of composite stiffened panels under compressive loads.

The initial defects during the manufacturing process are one of the main sources of uncertainties in estimating the strength of marine composite structures. Initial defects come in the form of initial delamination, impacted areas, and initial geometrical imperfections. Initial geometric imperfections could decrease the strength of composite stiffened plates. They refer to differences between the configurations of manufactured structures and the ideal design structures. These imperfections are due to the manufacturing process and are impossible to predict. Thus, different methods have been applied to model their effects on the strength of structures.

The postbuckling behavior of stiffened plates and that of unstiffened plates differ significantly. In stiffened plates, the plate panels are surrounded by stiffeners. Hence, their

boundary conditions are neither rotationally fixed nor free. Moreover, the stiffeners usually are designed to buckle after the plates; following plate buckling, the load then transfers to the stiffeners. The ultimate strength of stiffened plates is considerably greater than that of unstiffened plates. A large and growing body of literature has focused on the effects of imperfections on the strength of unstiffened rectangular/square laminated plates. These studies have used analytical methods to solve the governing nonlinear equations for the postbuckling of laminated plates.

Stamatelos et al. (2011) and Bisagni and Vescovini (2009) proposed analytical methods to investigate the buckling and postbuckling strength of stiffened isotropic and orthotropic plates. The former research group modeled stiffeners by using transverse and rotational springs with variable stiffness, while the latter modeled stiffeners by using De Saint Venant torsion bars. They both extended a model to analyze the postbuckling of perfect stiffened panels in local buckling mode with no damage. Ghannadpour and Barekati (2016) solved the laminated plate equation by using equivalent finite double Chebyshev polynomials. Karrech et al. (2017) presented a method on the basis of energy conjugacy to analyze thick plates' postbuckling behavior with large geometric imperfections. As they used the Fourier series to solve the governing equations, their method covers a wide range of imperfection shapes.

In analyzing the effects of geometric imperfections on the postbuckling behavior of unstiffened composite plates, many researchers have defined geometric imperfections by using sine or cosine functions. In many analytical solutions, a plate deformation domain is also defined using these functions, and this coordination can facilitate the analysis process. Moreover, many semi-analytical methods for studying the postbuckling of unstiffened rectangular/square laminated panels are available in the literature. In existing studies, the nonlinear large deflection plate equations are solved for various loading and boundary conditions.

To solve the governing equations of imperfect composite plates, Ghannadpour and Shakeri (2018, 2020) presented a new energy-based collocation method. They discretized the plate region with Legendre–Gauss–Lobatto nodes and used Legendre basis functions for displacements. In this method, the progressive failure of composite plates is also modeled. They proposed three methods to model material degradation. Ghannadpour and Abdollahzadeh (2020) developed a semi-analytical method to solve the governing nonlinear equations for thick laminated plates under in-plane compression. They applied the Hashin–Rotem failure criteria alongside two degradation methods and first- and higher-order shear deformation theories.

Semi-analytical methods are more versatile than analytical methods as the initial geometric imperfection shapes could be increasingly complicated. In using semi-analysis

methods to solve nonlinear composite plate equations, the effects of progressive failure can also be considered. Another method to analyze the imperfect plate strength is the finite stripe method, in which a plate is discretized into stripes. Mittelstedt and Schroder (2010) and Mittelstedt et al. (2011) solved Marguerre-type equations for imperfect composite plates under longitudinal compression and in-plane shear by using the Galerkin method. They derived a simple closed-form solution for the postbuckling of longitudinally compressed imperfect composite plates. Assaee et al. (2012) presented a semi-energy finite strip formula to analyze the nonlinear postbuckling of an imperfect laminated composite plate with a longitudinal sinusoidal imperfection shape.

Over the last decade, the buckling and postbuckling of unstiffened panels have attracted considerable attention. However, to the best of the authors' knowledge, the studies on the effects of initial geometric imperfections on the postbuckling behavior of stiffened composite plates are scarce. Elseifi et al. (1999) proposed a convex model to evaluate the effects of imperfections on the strength of stiffened plates. They used a linear combination of buckling mode shapes to find the worst-case scenarios for geometric imperfection shapes. This method stipulates that the first derivative of a stiffened plate's ultimate strength concerning each buckling mode amplitude is constant. This assumption is only valid if the initial geometric imperfections are small. In plates with large geometric imperfections, membrane stress develops along with bending stress. Hence, the form of deformation and the buckling and postbuckling behaviors of these plates are different from those of plates with small initial deformations.

Ambur et al. (2004) used the measured imperfection of a manufactured composite stiffened plate in a finite element model and calculated the postbuckling of the stiffened composite plate. Although this method is very accurate, it is almost impossible to use in estimating the strength of imperfect stiffened plates. Specifically, this method requires one or more full-scale samples of stiffened plates. The plates' geometric imperfections are then measured, and the measurements are used in a finite element model to calculate the strength. More importantly, this method only considers one case of many possible initial geometric imperfection shapes.

Anyfantis and Tsouvalis (2012) assumed that a geometric imperfection carried the form of the first buckling mode shape and investigated the postbuckling behavior of stiffened composite plates. They assumed that the magnitude of the imperfection is 1% of the thickness. Morshedsolouk and Khedmati (2014) and Smith and Dow (1985) used the $w^0 = \sum_1^n W_{mn} \sin \frac{\pi x}{a}$ function to model the imperfections of stiffened composite plates. Anyfantis (2019) studied the effects of local and global geometric imperfections on a stiffened panel's buckling load. The one-factor-at-a-time method (OFTM) and 2^k factorial design method were employed to

analyze the results. The significant modes or mode combinations for the buckling of conventional steel marine stiffened panels were specified. The author concluded that in analyzing the postbuckling strength of steel marine stiffened panels, global imperfections and stiffener imperfections are of little importance. Gaitanelis et al. (2019) proposed a method to analyze the in-plane compressive strength of an impacted composite stiffened panel. They used a user-defined progressive damage code in Abaqus in parallel with cohesive zone elements. In their study, they assumed the first eigenvalue buckling mode as the initial geometric imperfection.

Given the lack of research in this field, the present study investigates the effects of initial geometric imperfections on the postbuckling behavior and ultimate strength of marine stiffened composite plates. Given the absence of an approved systematic way to analyze the effects of imperfections on the postbuckling of stiffened composite plates, the best practice is to use the experience and methods in the field of steel structures. Thus, this work adopts the suggestion of EN 1993-1-5, which was also used by Tran et al. (2014). The geometrical nonlinear finite element method is combined with progressive failure analysis and used to study the effects of initial geometric imperfections on the postbuckling behavior, ultimate strength, and load-carrying capacity of a marine composite stiffened plate. This study provides insights into the problem of imperfect stiffened composite plates. Moreover, the magnitude of imperfection is noted as the maximum initial deformation of the stiffened plate in the y -direction.

2 Finite Element Modeling

Recent advances in nonlinear finite element methods enable us to study the consequences of geometric imperfections in structures at a relatively low cost. As the thickness of stiffened composite plates used in shipbuilding is relatively small, these materials can be modeled using shell elements. Large deflections and composite material fractures due to buckling and progressive failure are considered.

In this study, a composite stiffened plate with practical dimensions applicable to seagoing composite ships is chosen from the study of Chen and Guedes Soares (2008a, b). The stiffened panel measures 1000 mm long and 456 mm wide, and the span of its stiffeners is 300 mm (Figure 1). The panel is stiffened with two hat-type stiffeners, the cross-section dimensions of which are shown in Figure 2. The problem is solved using the displacement-controlled method. The displacement in the x - and y -directions of line CD is assumed to be zero, and the displacement in the z -direction of line CD is increased incrementally. The nodes on line AB are fixed. Moreover, the rotation on lines

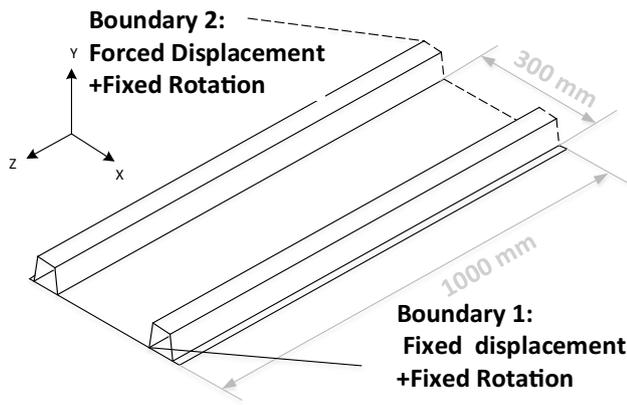


Figure 1 Geometrical dimensions and boundary conditions of the model

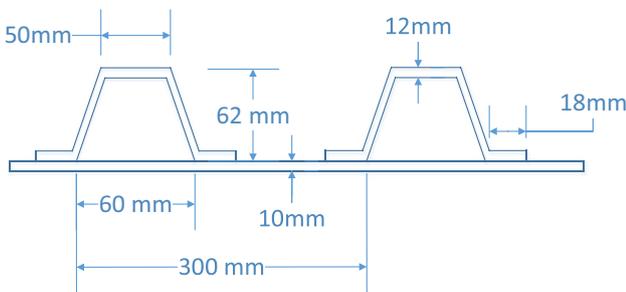


Figure 2 Cross-section dimensions

AB and CD is constrained. These boundary conditions are expected to produce longitudinal in-plane compression in the *z*-direction.

The plate and its stiffeners are made of E-glass epoxy woven material that is commonly used in marine applications. Their material properties are presented in Table 1. The stacking sequences of the plate panels and stiffeners

are $[0_4/45/0_4/45]_S$ and $[0_5/45/0_5/45]_S$, respectively, given the *z*-axis in Figure 1.

To simulate the postbuckling of the composite stiffened panels, this study uses progressive failure analysis and geometrical nonlinear properties. Murugesan and Rajamohan (2017) divided the progressive failure analysis method into three steps: analysis of laminate for stress distribution, application of an appropriate failure criterion, and application of material or stiffness degradation methods. To implement these steps, the present study conducts postbuckling analysis by using Workbench ANSYS 18.1. The modeling and analysis steps are illustrated in Figure 3 and comprise the following phases:

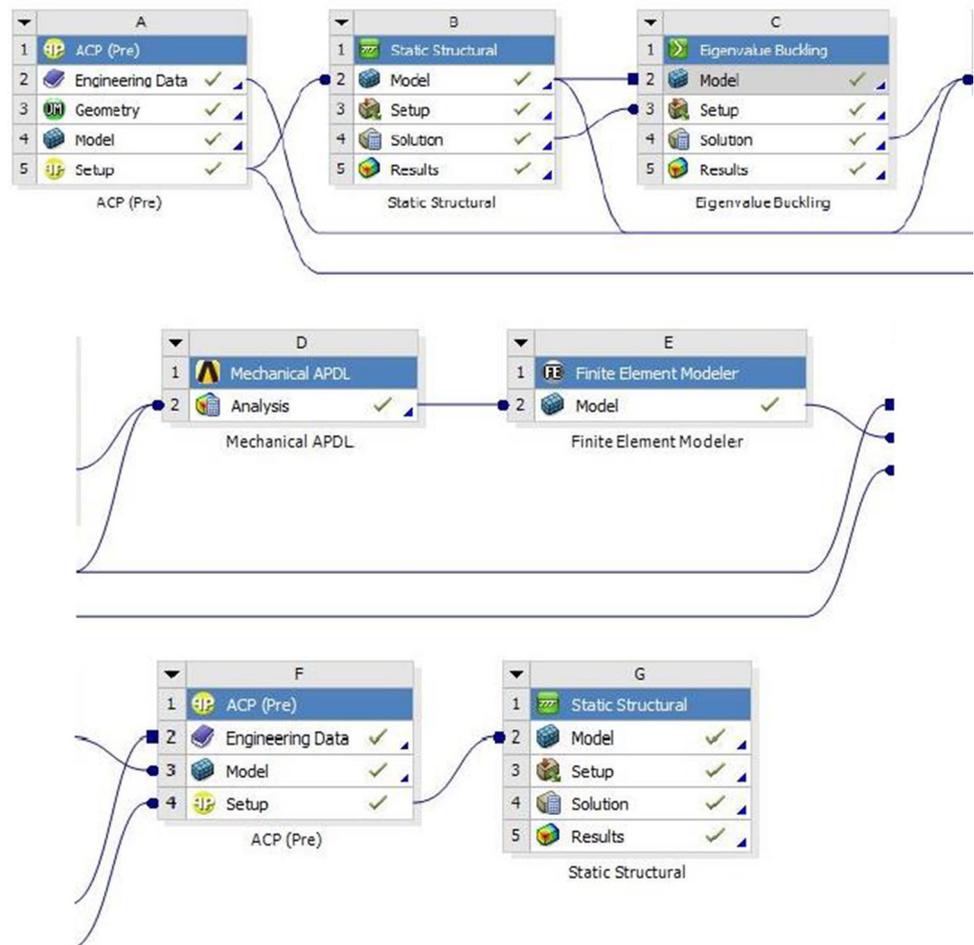
- 1) Modeling the composite stiffened plate in the ACP (pre) component system (ACP or Ansys Composite PrepPost is an add-on module for ANSYS for modeling layered composite structures);
- 2) Defining the boundary conditions in the static structural analysis system, with the buckling modes derived on the basis of the loading conditions defined in this step;
- 3) Deriving the first 11 buckling modes in the eigenvalue buckling analysis system;
- 4) Applying the initial geometric imperfection in the mechanical APDL component system (using UPGEOM command and saving the FEM model); in this study, the initial geometric imperfection defined herein on the basis of the maximum initial deformation in the stiffened plate;
- 5) Preparing the FEM model in the finite element modeler component system;
- 6) Redefining material properties in the ACP (pre) component system; note that the ACP model cannot be transferred into the mechanical APDL entirely and that only the geometry can be transferred and changed; the properties should be defined again to generate the composite material model;

Table 1 Mechanical properties of the material used in this study

Mechanical properties	Symbols	Magnitude
Module of elasticity in the main direction of the material	E_1 (GPa)	15.8
Module of elasticity in a direction normal to the main direction of the material	E_2 (GPa)	15.8
Shear modulus in directions 12 and 13	G_{12} and G_{13} (GPa)	3.5
Shear modulus in direction 23	G_{23} (GPa)	0.35
Poisson's ratio in directions 12 and 13	ν_{12} and ν_{13}	0.13
Tensile strength in direction 1	X_T (MPa)	249
Compression strength in direction 1	X_C (MPa)	213
Tensile strength in direction 2	Y_T (MPa)	249
Compression strength in direction 2	Y_C (MPa)	213
Shear strength	S (MPa)	23.5

Direction 1 is the laminate's main direction, and direction 2 is the direction normal to direction 1

Figure 3 Project schematic in Workbench ANSYS 18.1



- 7) Conducting nonlinear progressive failure analysis of the composite stiffened plate in the static structural analysis system; progressive failure analysis is performed using Hashin's failure criteria (Hashin 1980).

2.1 Buckling Mode Shape as Initial Geometric Imperfection

Initial geometric imperfections in composite structures have irregular stochastic local or global shapes. However, they are commonly modeled using buckling mode shapes (Anyfantis and Tsouvalis 2012; Anyfantis 2019; Gaitanelis et al. 2019; Tran et al. 2014). In this study, a sensitivity analysis is performed to study the effects of geometric imperfections on the postbuckling of the composite stiffened plates. The imperfection shapes are based on EN 1993–1-5, as described by Tran et al. (2014) and Anyfantis (2019). They have used four different imperfection shapes in their study, i.e., global buckling mode, local plate panel buckling mode, stiffener torsional buckling mode, and a set of combinations of these modes.

2.2 One-Factor-at-a-Time Method

The first step in understanding the effects of imperfections on stiffened plate strength is the implement the OFTM. In this method, the effect of each imperfection mode is analyzed one by one. Herein, the 1st to 11th buckling mode shapes are considered as initial geometric imperfections, and they are known as the 1st to 11th imperfection modes (Figure 4). The four first modes can be concluded as the local plate buckling mode, while the other modes are combinations of global and local modes.

2.3 Method of Combined Modes

After analyzing the panels' postbuckling ultimate strength with consideration of all imperfection shapes in the previous section, the maximum in-plane compressive load is illustrated vs. maximum geometric imperfection (modes 1 to 11). Then, the three worst cases are chosen (modes 11, 9, and 7). These initial geometric imperfection modes exert the most effect on the ultimate strength of the stiffened composite plates. In addition to all the considered buckling mode

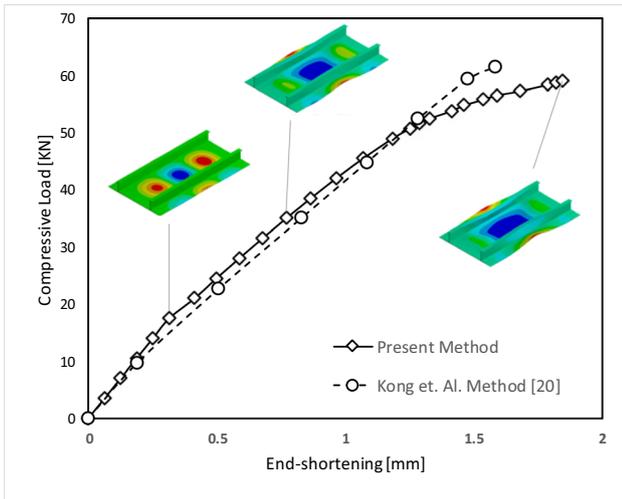
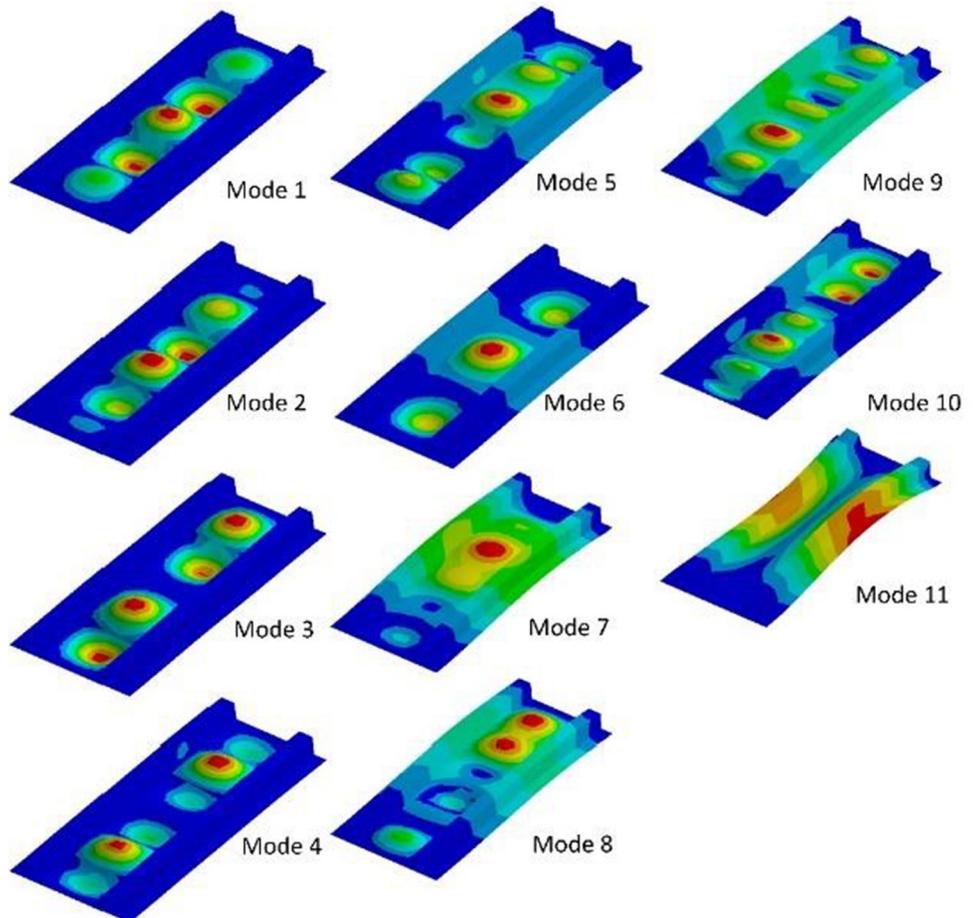


Figure 4 First to 11th imperfection modes

shapes, the effects of a set of linear combinations of these three worst cases as initial geometric imperfections on the stiffened plate’s ultimate strength are studied. These combination sets are.

Figure 5 Compressive load-end shortening for composite T-stiffened plate



- Combination model A1: $1.0 \times (\text{no. } 11) + 0.7 \times (\text{no. } 9) + 0.7 \times (\text{no. } 7)$
- Combination model A2: $0.7 \times (\text{no. } 11) + 1.0 \times (\text{no. } 9) + 0.7 \times (\text{no. } 7)$
- Combination model A3: $0.7 \times (\text{no. } 11) + 0.7 \times (\text{no. } 9) + 1.0 \times (\text{no. } 7)$

Here, no. i is the i th normalized mode shape.

3 Verification of the Numerical Modeling

To validate the proposed method, this work compares the corresponding results with those of Kong et al. (1998). A test was performed to model and analyze the strength of the composite stiffened plate with T-stiffeners. The details of the model and the loading and boundary conditions are given in Kong et al. (1998). The compressive load-end shortening diagram of this stiffened composite panel is shown in Figure 5. Under the in-plane compressive load, the plate panel first buckles. Meanwhile, with increasing compressive load, the force on the stiffeners increases. In the next level, the stiffeners show flanging, followed by torsional buckling.

4 Results and Discussion

In the present study, a series of nonlinear finite element models with progressive failure are explored to investigate the effects of imperfection modes and magnitude on the ultimate strength of a stiffened composite plate. The boundary and loading conditions are illustrated in Figure 1. The ultimate compressive strength of the imperfect stiffened composite plate versus the magnitude of the imperfection for all considered imperfection modes is plotted in Figure 4. In the perfect plate, the ultimate compressive strength is 1.08 MN (average stress of 216 MPa). The plate strength decreases with increasing initial imperfection. The rate of this reduction in ultimate strength depends on the initial imperfection modes. In this regard, the initial imperfections can be divided into three categories:

- 1) Local imperfections, in which only the plate panels are deformed, i.e., modes 1 to 4 (the results are shown as dashed lines in Figure 6);
- 2) Global imperfections, in which the stiffeners deform globally, i.e., modes 5 to 11 (the results are shown as bold lines in Figure 6);
- 3) Combination of three worst cases, i.e., A1, A2, and A3 (dotted lines in Figure 6)
- 4) In this figure, the results of cases A1 and A2 are very similar; therefore, the results are shown as one line.

When the magnitude of the imperfection is 20 mm and the imperfection is in the local mode, the ultimate compressive strength of the stiffened composite plate shows a 13% drop. Meanwhile, even a small imperfection in the global mode can decrease the ultimate strength by more than 13%.

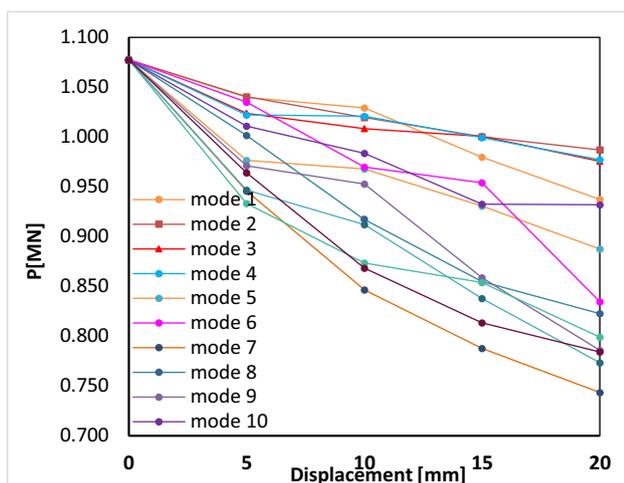


Figure 6 Maximum in-plane compressive load — maximum geometric imperfection

In the worst case among the global modes, the strength is reduced by 31% for a 20-mm imperfection. In marine structures, stiffeners buckle after plate buckling and thus carry the resulting loads. Hence, stiffeners' strength and their imperfections exert a considerable effect on stiffened plates' ultimate strength.

In this study, various geometric imperfection shapes are considered. For each imperfection shape, various magnitudes are studied. In this section, some cases are explored in detail.

4.1 Case 1: No Imperfection

For the perfect stiffened composite plate, the results of the finite element analysis are shown in Figure 7. The horizontal axis represents the longitudinal deformation of boundary 2, called Δ (Figure 1), and the vertical axis is the applied longitudinal load. In Figure 7, the behavior of the stiffened composite plate is initially linear, and all structural components are in the linear elastic region. When the first ply failure occurs in the layers with a 45° fiber angle, it quickly progresses and spreads to all layers. Following these failures, the stiffness of the stiffened composite plate and the P- Δ curve slope decreases. By increasing the forced displacement on line CD thereafter, the behavior of the stiffened composite plate becomes bilinear until the plate panel buckles in 5 half-waves (first eigenvalue buckling mode), and the stiffness of the plate decreases again. Subsequently, the plate reaches its ultimate strength, which is equal to 1.08 MN. Increasing the forced displacement at boundary 2 eventually causes the stiffened composite plate to become unstable following the stiffeners' buckling.

The deformation and damage propagation in the first and fifth layers of the plate panels and the first and sixth layers of the stiffeners for the perfect stiffened composite plate are

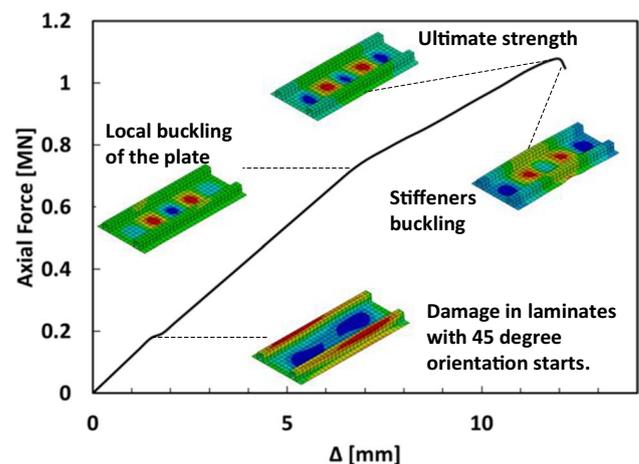


Figure 7 Longitudinal compression load versus longitudinal displacement for perfect stiffened composite plate

presented in Tables 2 and 3, respectively. These layers are chosen as examples and as representatives of the other layers because of their similar behaviors.

The first and fifth layers are chosen as examples, and they can represent the behaviors of the other layers. In the perfect composite stiffened plate, when the displacement Δ reaches 5.98 mm, the first failure in the matrix of layer 1 occurs. This damage starts in the middle of lines AB and CD (Figure 1). Thereafter, the damage spreads fast along both boundaries. Then, at $\Delta = 7.06$ mm, the

matrix damage at the middle of the plate starts to grow. At $\Delta = 7.8$ mm, fiber damage initiates at the same place. Eventually, as the fiber damage progresses to the sides, the stiffened composite plate reaches its ultimate strength. In layer 5, whose fiber orientation is equal to 45° , the matrix damage at $\Delta = 1.3$ mm initiates at the corners of the plate at the boundaries and progresses to the whole plate at a high rate. In layer 5, fiber damage also occurs in small regions before the plate reaches its ultimate strength. Table 4 illustrates the deformation and damage

Table 2 Deformation and progress of damage in layers 1 and 5 in plate panels. 0 means no damage, 1 means initiation of damage, 2 means complete damage

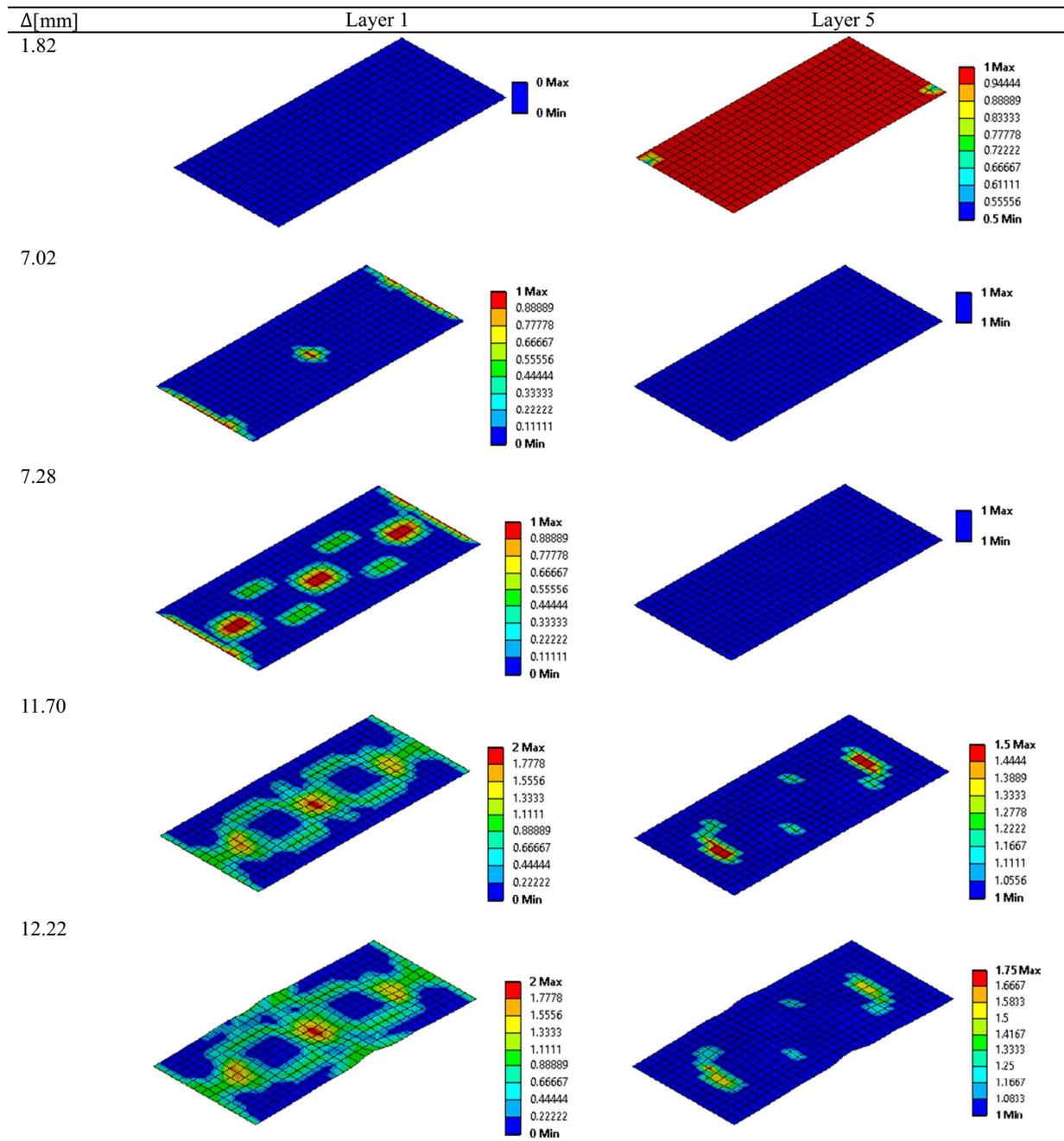
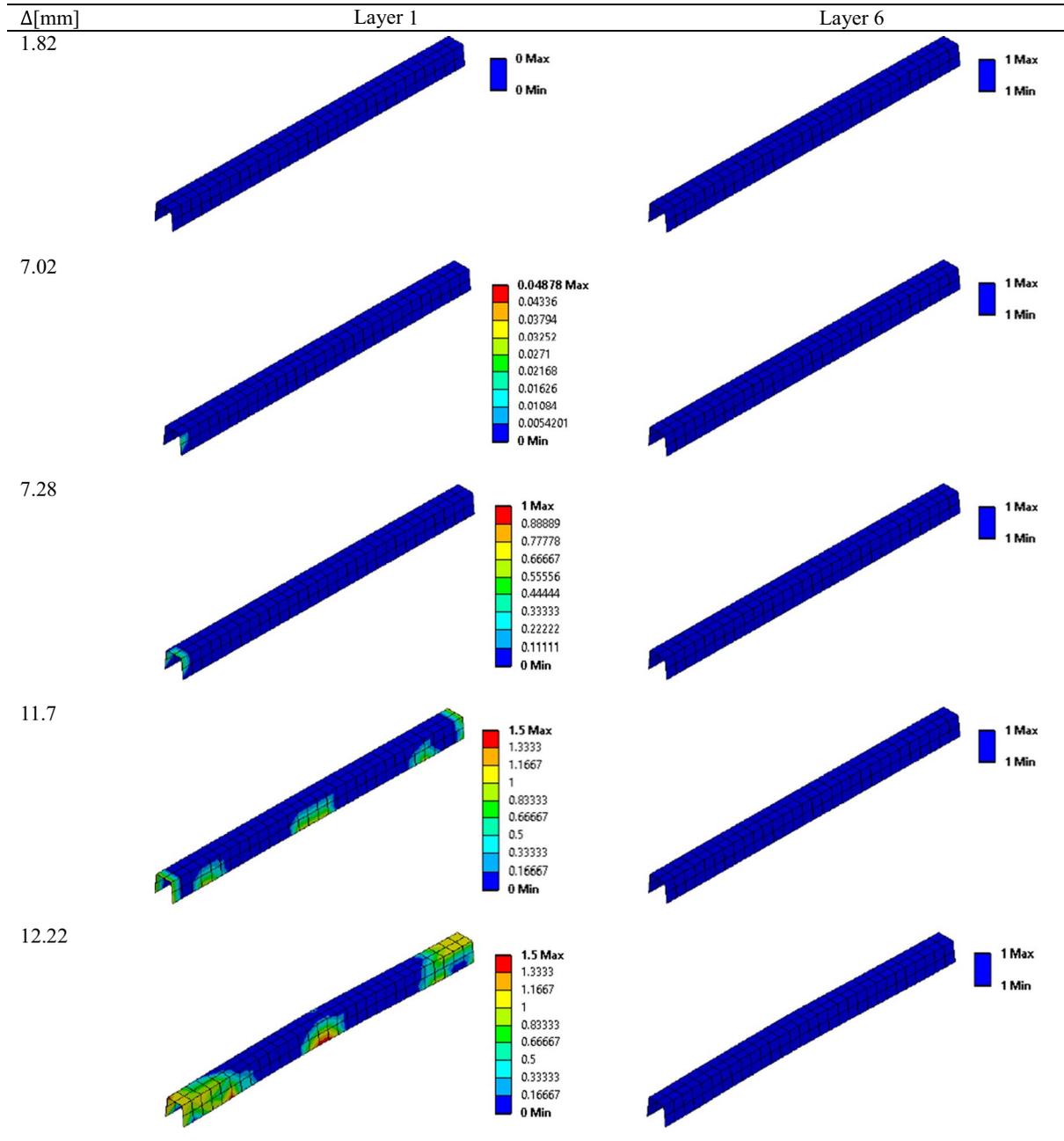


Table 3 Deformation and progress of damage in layers 1 and 5 in the stiffeners. 0 means no damage, 1 means initiation of damage, 2 means complete damage



propagation in layers 1 and 6 of the stiffeners of the stiffened composite plate with no imperfection. In this case, in stiffener layer 1 at $\Delta = 7.5$ mm, the matrix damage starts in the middle and near the stiffeners' boundaries in the webs. By increasing the displacement to $\Delta = 11.44$ mm, the fiber damage starts in the middle of the stiffeners in the webs. Web damage also takes place in the webs and boundaries. In layer 6 of the stiffeners, the matrix damage in the boundaries at $\Delta = 1.56$ mm expands to the entire layer at a high rate; at $\Delta = 12.09$ mm, fiber damage is initiated in the middle of the stiffeners' webs.

4.2 Case 2: Imperfection Mode 11 with a Magnitude of 20 mm

In this case, the model and material properties are the same as those in case 1, while the initial geometric imperfection has the shape of the 11th buckling mode, and its magnitude is equal to 20 mm. The results for the 11th mode of imperfection with different magnitudes are presented in Figure 7. The magnitude of the imperfection exerts a dramatic effect on the postbuckling behavior of the stiffened composite plate. The imperfection magnitude of 20 mm of the 11th

Table 4 Damage sequence in stiffened plate

Δ (mm)	Layer no.	Plate/stiffeners	Fiber/ Matrix damage	Damage location
1.3	5th	Plate	Matrix	Corners of the plate
1.56	6th	Stiffeners	Matrix	Boundaries
5.98	1st	Plate	Matrix	Middle of boundary 1 and boundary 2
7.06	Local buckling of the plate			
7.06	1st	Plate	Matrix	Middle of the plate
7.50	1st	Stiffeners	Matrix	Middle and near of the boundaries in the webs
7.80	1st	Plate	Fiber	Middle and near of the boundaries in the webs
12.09	6th	Stiffeners	Fiber	Middle of the webs
12	5th	Stiffeners	Fiber	Middle of the stiffeners

mode is chosen to study in detail the imperfection effect on buckling and postbuckling.

Different from the case of the perfect plate, the buckling mode of the plate under the 11th imperfection mode with a magnitude of 20 mm (Tables 5 and 6) is not apparent. The damage in the plate and stiffeners starts with matrix damage in the layers with a fiber orientation of 45° at $\Delta = 0.5$ mm, and the damage spreads through these layers rapidly. Consequently, the slope of the P- Δ curve (Figure 8) decreases. In other layers, matrix damage starts at $\Delta = 3.5$ mm in the middle part of the plate and boundaries.

Meanwhile, fiber damage starts in the layers with a fiber orientation of 0° at $\Delta = 7.675$ mm and spreads to the adjacent layers at $\Delta = 9.7$ mm. Then, at $\Delta = 7.15$ mm, the fiber damage in the boundaries of the stiffeners starts. At $\Delta = 9.325$ mm, fiber damage begins at the middle of the stiffeners' table in layers with a fiber orientation of 0° and spreads to the stiffeners' webs. At $\Delta = 9.7$ mm, this damage spreads to all the layers in the middle of the table. When

the stiffener fibers in the stiffeners' table and webs fail, the stability of the stiffened composite plate is lost. Comparing these two cases shows that initial geometric imperfection modes and magnitudes can change the ultimate strength and postbuckling behavior of stiffened composite plates. This result should be considered in the analysis of the strength of these structures.

5 Conclusions

In this work, a series of nonlinear finite element and progressive failure was performed on a stiffened composite plate with a wide range of initial geometric imperfections. The plate is subjected to an in-plane compression load. For all imperfection cases, the P- Δ curves are derived, and the effects of different imperfections are studied. The results can be summarized as follows:

- 1) All the P- Δ charts suggest that designing a stiffened composite plate on the basis of the first ply failure is not accurate and underestimates the strength of the plate. In all cases, the stiffened composite plate's ultimate strength exceeds the first ply failure's strength (Figure 6).
- 2) The initial imperfection can be categorized into two groups, namely, local and global modes. In the local mode of initial geometric imperfections, the stiffeners are in perfect shape, and only the plate panels are deformed. In the global mode, the stiffeners show bending or torsional deformation. These results indicate the following:
 - a A 20-mm local geometric imperfection can reduce the stiffened plate's ultimate strength by 13%;
 - b A 20-mm global geometric imperfection can decrease the stiffened plate's ultimate strength by 31%.

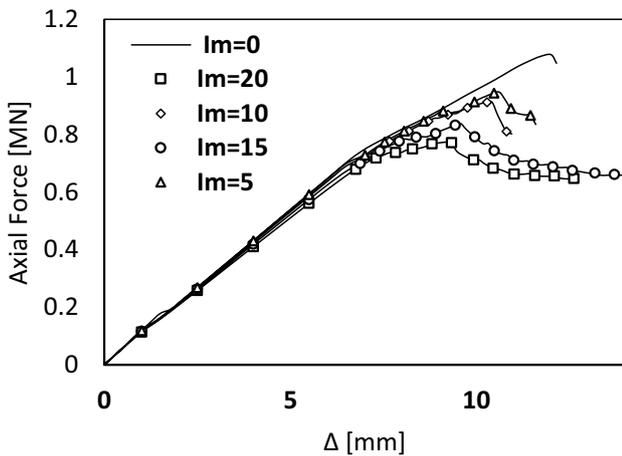


Figure 8 Longitudinal compression load versus longitudinal displacement for mode 11, I_m is the magnitude of the imperfection

- 3) Global mode shapes can decrease the ultimate strength of stiffened plates dramatically. For example, all the six worst cases herein, including modes 7, 8, 9, and 11, show global mode shapes.
- 4) In addition to buckling mode shapes, three sets of combinations of the worst cases are studied. If a geometrical imperfection shape is a combination of three buckling mode shapes, then the postbuckling behavior of the plate under such combination is almost the same as that in the global mode.
- 5) The postbuckling behavior of the plate highly depends on the plate imperfection mode, especially for large imperfections. Thus, we highly recommend that the initial geometric imperfections in the global mode be investigated during the design phase and be limited during the manufacturing and surveying phases of the production of stiffened composite plates.

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