

Effect of Injection Angle on Artificial Cavitation Using the Design of Experiment Method

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Abstract: Using the supercavitation phenomenon is necessary to reach high velocities underwater. Supercavitation can be achieved in two ways: natural and artificial. In this article, the simulation of flows around a torpedo was studied naturally and artificially. The validity of simulation using theoretical and practical data in the natural and artificial phases was evaluated. Results showed that the simulations were consistent with the laboratory results. The results in different injection coefficient rates, injection angles, and cavitation numbers were studied. The obtained results showed the importance of cavitation number, injection rate coefficient, and injection angle in cavity shape. At the final level, determining the performance conditions using the Design of Experiment (DOE) method was emphasized, and the performance of cavitation number, injection rate coefficient, and injection angle in drag and lift coefficient was studied. The increase in injection angle in the low injection rate coefficient resulted in a diminished drag coefficient and that in the high injection rate coefficient resulted in an enhanced drag coefficient.

Keywords: injection angle, supercavitation, artificial cavitation, torpedo, design of experiment, drag coefficient, lift coefficient

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1 Introduction

The super cavitation process should be used to reach high velocities underwater. This phenomenon can be achieved naturally and artificially. Bulbs are usually generated by a device called a cavitator, which is placed at the tip of the vehicle and implemented especially for this purpose. Cavitators come in different types, including conic, conical, and disc shape (Goel, 2002). Cavitation streams are expressed as $\sigma_c = \frac{P_\infty - P_v}{1/2\rho V_\infty^2}$, where P_∞ and V_∞ are absolute

pressure and relative velocity of a water stream with a torpedo, respectively, P_v is the steam pressure in the peak temperature of liquid, and ρ is the liquid density.

The use of a cavitator may not be sufficient to create cavitation. Therefore, air is blown at the tip of the different parts of the body of a vehicle to continuously generate cavitation (Goel, 2002). The amount of air with a gas injection

rate coefficient $q_g = \frac{Q_g}{V_\infty D_n^2}$, where Q_g is the gas volume flow, and D_n is the cavity diameter.

Cavitation streams are dynamic and complex. The smallest change in the stream field affects the shape of the bulb and its parameters. As it also influences the prediction and development of the bulb, the numerical and experimental research in the cavitation field has been conducted broadly. Previous literature investigated the changes in different parameters, including cavitator shape, attack angle, and gas injection rate coefficient, among others, and analyzed their effects on natural and artificial cavitation bulbs. On the basis of previous studies, we consider that to investigate under-water vehicles, such as torpedoes, the following points should be regarded in the model:

- The existence of a body at the back of the cavitator should be considered as the body behind the cavitator affects the shape and symmetry of the bulb (Schauer, 2003; Ma *et al.*, 2006; Alishahi, 2010).
- The effect of gravity should be observed (Zhang *et al.*, 2007).
- If the body is symmetric, a good correspondence will be achieved between 2D and 3D results, and the return and hydro pulse leakages will be visible (Ahn *et al.*, 2012).
- The number of natural cavitation affects the bulb shape made out of gas injection; therefore, these two phenomena should be investigated together (Yang *et al.*, 2009; Wei *et al.*, 2009).
- The artificial supercavitation is a timely phenomenon (Zou *et al.*, 2010; Vlasenko and Savchenko, 2012)

One of the most important experimental studies conducted in recent years is Schauer (2003). We used Schauer's model in the present study because the details of Schauer's study are completely available. Moreover, the model enables us to examine artificial and natural cavitation separately and simultaneously, which is one of the purposes our study, and to investigate artificial supercavitation in low velocity in torpedoes. In the current research, a numerical study was implemented in two dimensions, and the change in injection angle in artificial cavitation and the effect of this change on the field of the speed of the stream and the bulb shape were investigated. A side from the parameters mentioned by (Wei *et*

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al., 2009), the gas injection angle is also an important factor influencing the shape of the bulb. Therefore, the three parameters of injection rate coefficient, injection angle, and cavitation number can affect on the drag and lift coefficients. Through the Design of Experiment (DOE) method, we defined the practical conditions of the experiment. The DOE was implemented to define the practical conditions. The relations of the objective functions of the lift and drag coefficients were calculated according to injection angle, injection rate coefficient, and cavitation number, and the related charts were presented.

2 Numerical simulation

In this numeral solution, the non-linear Reynolds averaged Navier-Stokes equations and the auxiliary equation for the volume fraction of water are solved based on pressure completely and implicitly. A simple algorithm was used for the dependency of the velocity and pressure field (Baradaran Fard and Nikseresht, 2012). Non-temporal resolution was used for natural cavitation and temporal resolution was used for artificial cavitation (Alishahi, 2010). A mixed model was used to demonstrate the multi-phase flow (Hashem Abadi and Dehnavi, 2011). The turbulence model of the mixture is a multi-phase turbulence model as pre-supposed. This model is the extension of the single phase $k-\epsilon$ model. In this situation, taking advantage of the mixture features and the mixture velocities to determine the turbulence stream characteristics is sufficient. The geometry of the torpedo is designed based on the geometry implemented in Schauer's experiment (Schauer, 2003) in which the cavitator diameter is 10 mm, the diameter of the torpedo body is 16 mm, and the length of the torpedo is 107 mm.

Fig. 1 shows the circumference and the solution borders surrounding the torpedo. Regarding the asymmetry of the bulb in the presence of the acceleration of gravity around the torpedo body, the axial symmetry of the shape is ignored (Zhang et al., 2007).

The upper border of the stream is located at $30D_n$, i.e., 300 mm away from the cavitator, and the downstream border is located 700 mm away from the cavitator. The upper and lower walls are 100 mm away from the center line (Ji et al., 2010). The upper border of the velocity stream at the entrance and the downstream pressure at exit in the torpedo body is the pre-condition for non-slipping. For air injection in artificial cavitation, entering velocity in the injection point and in the upper and lower walls are regarded as the pre-conditions of freely slipping.

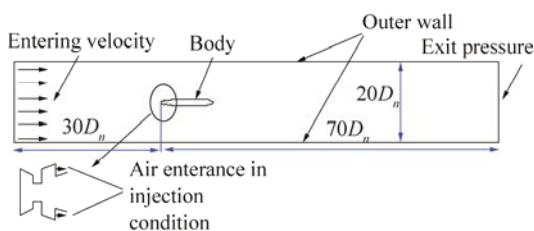
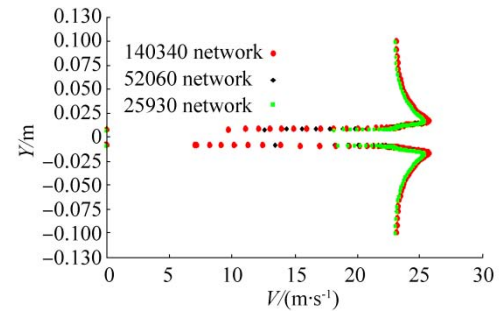


Fig. 1 Solution borders

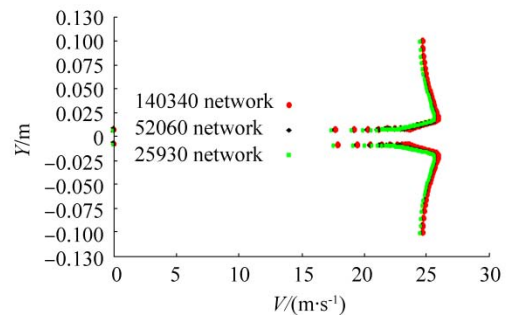
2.1 Independence from networks

For the stream parameters to be independent from a number of networks, the ideal network for the solution field should be chosen in a way that it does not increase the calculations and does not introduce errors into the simulation.

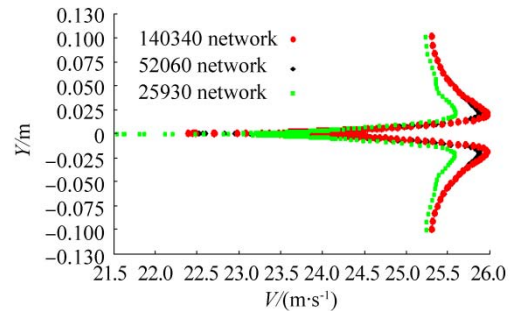
In Fig. 2, the velocity profiles for the three networks in the three cross-sections with distances of 3, 9, and 15 cm at the back of the cavitator are shown. As defined in the figure, the changes in the velocity profile in two networks, namely, 52 060 and 140 340, are close to each other. As a result, network 52 060 is independent from network changes.



(a) Velocity profile in $x = 0.03$ m



(b) Velocity profile in $x = 0.09$ m



(c) Velocity profile in $x = 0.15$ m

Fig. 2 Analysis of the independence of the velocity profile network in three cross-sections

2.2 Analyzing the network quality

We analyze Y^+ around the torpedo body to analyze the quality of the selected network. To attain this aim, Y^+ should be less than 100 (Ji et al., 2010, Roohi et al., 2013). Y^+ is a non-dimensional distance. It is often used to describe how coarse or fine a mesh is for a particular flow pattern. It is important in turbulence modeling to determine the proper size of the cells near domain walls. The turbulence model wall laws have restrictions on the Y^+ value at the wall. For instance, the standard K-epsilon model requires a wall Y^+

value between approximately 300 and 100. A faster flow near the wall will produce higher values of Y^+ , so the grid size near the wall must be reduced.

As shown in Fig. 3, the amount of Y^+ is below 60. Therefore, network 52 060 is selected as the optimal network for simulation. Fig. 4 shows network 52 060 along the edges of the shape in detail.

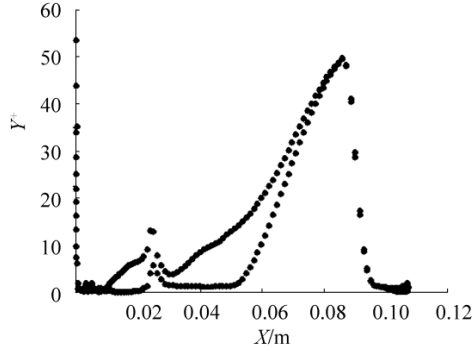
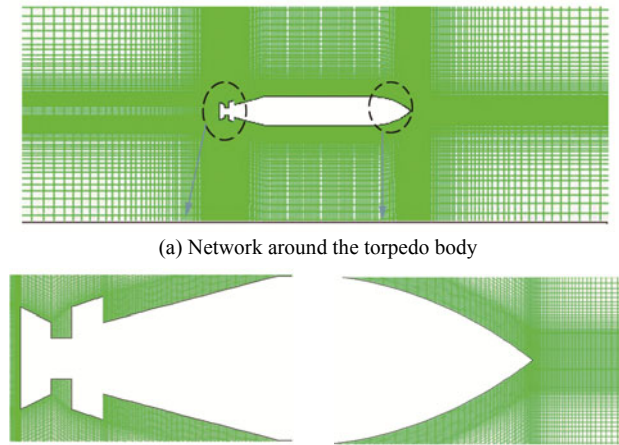


Fig. 3 Amount of Y^+ around the torpedo body in 52 060 network



(a) Network around the torpedo body (b) Close to tip view of the torpedo (c) Close to end view of the torpedo

Fig. 4 Selected network 52 060

3 Validation of artificial cavitation

Fig. 5 presents the results of artificial cavitation compared with those of the experimental one. To calculate the artificial cavitation number in terms of the length and diameter of the bulb, the semi-experimental wide relation is used. The wide relation for a disc cavitator with zero attack number is defined as follows (Schauer, 2003):

$$L = \frac{1.08}{\sigma_c^{1.118}} D_n \quad (1)$$

$$D = \frac{0.534}{\sigma_c^{0.568}} + 1 \quad (2)$$

where D_n is the cavitator diameter, $\sigma_c = \frac{P_\infty - P_c}{1/2 \rho V_\infty^2}$ is the artificial cavitation number, and P_c is the inside pressure of the bulb.

As shown in Fig. 5, the shape of the simulated bulbs matches well with the experimental results.

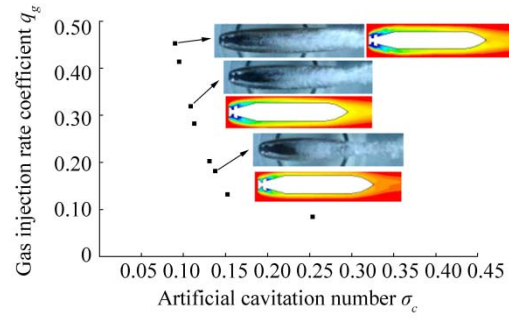


Fig. 5 Comparison of the simulation results with Schauer's laboratory results (Schauer, 2003)

4 Result and discussion

4.1 Effect of gas injection method

To examine the effect of the gas injection method on the air stream entrance border, we implement the angle condition to enable the preferred angle to enter the air stream. The injection angles in three situations are the 0° , 30° , and 60° angle changes.

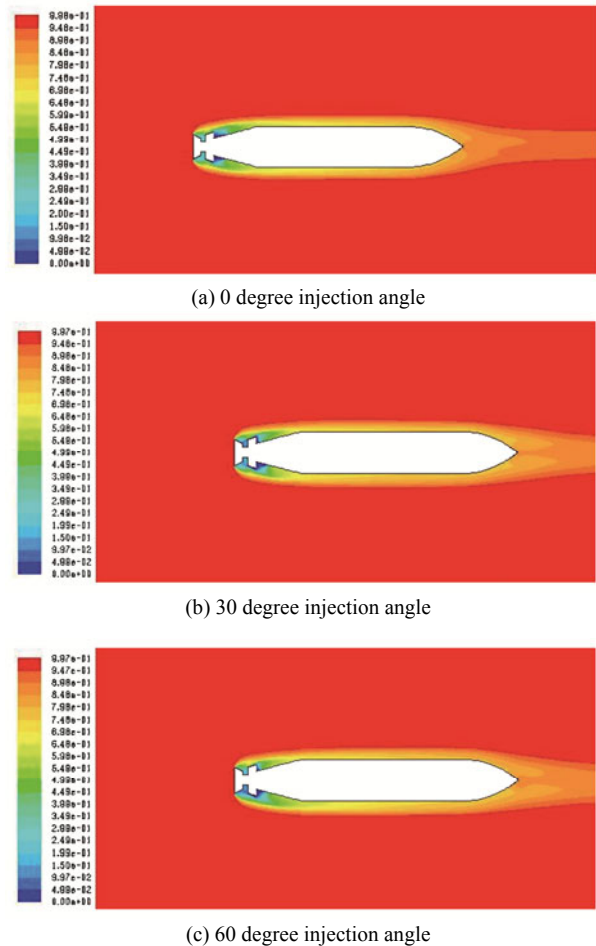


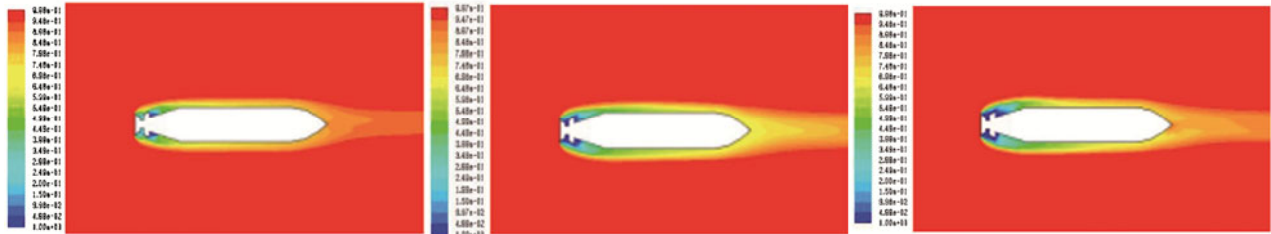
Fig. 6 Comparison of changes in the artificial cavitation phase in different injection angles at an injection rate coefficient of 0.18 and $\sigma_v=1$

As shown in Fig. 6, the increase in angle results in the

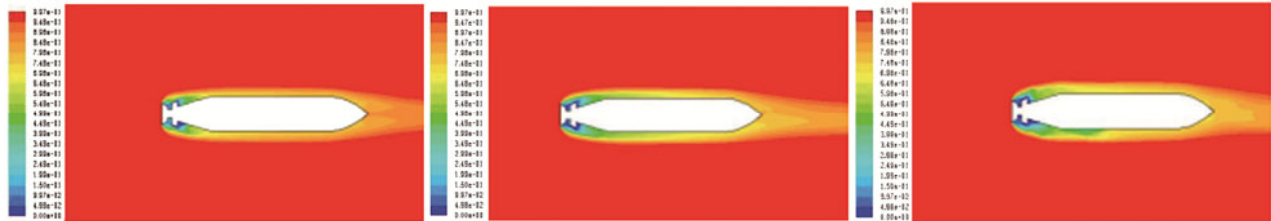
increase in volume of the bulb. To what extent these changes are useful and how other parameters (e.g., amount of the injection rate coefficient and number of natural cavitation) affect the artificial cavitation bulb in different angles should also be investigated.

In Fig. 7, phase counters are compared in three different injection angles and injection rate coefficients at $\sigma_v=1$. Phase counters are presented according to the increase in injection rate coefficients from right to left and according to injection angle from up to down.

0 degree injection angle:



30 degree injection angle:



60 degree injection angle:



(a) 0.18 injection rate coefficient

(b) 0.32 injection rate coefficient

(c) 0.45 injection rate coefficient

Fig. 7 Comparison of phase counters in three different injection angles and injection rate coefficients at $\sigma_v=1$

With increasing angle, the bulb length in all three injection rate coefficients, except the 0.32 injection rate and 60° injection angle, also increases. The shape of the bulb phase counter around the torpedo in the 0.18 injection rate coefficient and 30° injection angle is near the phase counter shape in the 0.32 rate coefficient and 0° injection angle. This fact applies to the shape of the bulb phase counter around the torpedo in the 0.18 injection rate coefficient and 60° injection angle. In other words, the bulb shape in the 0.18 injection rate coefficient gets closer to the bulb shape in the 0.32 injection rate coefficient with increasing injection angle. However, this outcome does not mean that the change in injection angle is useful 100%. The increase in injection rate coefficient causes pulsing cavitation in the 0.45 injection rate coefficient. This situation is observed in the 0.32 injection rate coefficient and 30° injection angle. The figure shows that the bulb in the 0.32 injection rate coefficient and 30° injection angle is near the bulb in the 0.45 injection rate coefficient and 0° angle. In the other words, the cavitation bulb may reach an inconsistent state with the increase in injection rate coefficient and injection angle. In the 0.45 injection rate coefficient, an

increase in injection angle causes an increase in the wave-like state in the bulb wall. It also disturbs the bulb symmetry to 0.5 volume fraction of the phase changes (i.e., the green color in 0.45 injection rate coefficient of the phase counter). An increase in bulb asymmetry disturbs the upside and the downside with an increase in injection angle. Up to this stage, variation analysis of the changes in cavitation number 1 is conducted. Cavitation number 1.5 is also simulated, and the results of phase counters are similar to those of cavitation number 1. As no steam is found in the steam phase, we do not mention the results. The situation in cavitation number 0.5 is different as some steam is observed. In this special condition, artificial and natural cavitations exist simultaneously together. Fig. 8 presents the comparison of the interaction of the two phases in the injection rate coefficients of 0.18 and 0.45, 0° injection angle, and $\sigma_v=0.5$. The increase in injection rate coefficient removes the steam phase caused by natural cavitation. In the 0.45 injection rate coefficient, an air injection penetrates the forward side of the torpedo and surrounds the whole torpedo body. This issue is investigated in a previous study (Ji et al., 2010).

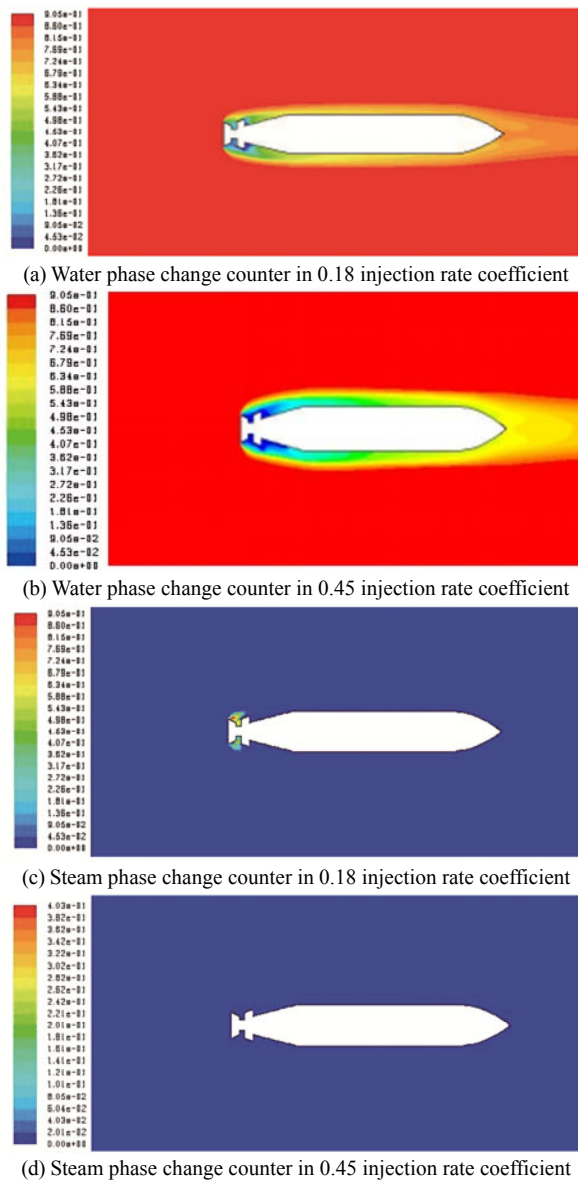


Fig. 8 Comparison of interaction between two phases in the 0.18 and 0.45 injection rate coefficients, 0° injection angle, and $\sigma_v=0.5$

5 Design of experiments

The shape of the bulb and its effects are shown in the drag and lift coefficients. Therefore, the general result is affected in terms of the three parameters of injection rate coefficient, injection angle, and cavitation number on the drag and lift coefficient. The test design is examined using the DOE method to determine the change in injection angle and effect parameters. The DOE method shows the parameter number and each parameter level in the target subject with a random test. The current study addresses the factorial design at three levels. Therefore, to observe the change in target subject and operation of the injection angle, the three parameters of cavitation number, injection angle, and injection rate coefficient are considered in three cases (Table 1). Cavitation number is for visible velocity and temperature (i.e., for

evaporation pressure), and water pressure is at three levels of 0.5, 1, and 1.5. Torpedo velocity is not high in the injection in primary moments when natural cavitation is achieved. Therefore, natural cavitation is considered for an amount when natural cavitation is not or is forming. The injection angle is considered at three logical levels of 0°, 30°, and 60° and the injection rate coefficient is considered at the three levels of 18%, 32%, and 45%. The injection rate coefficient in the bulb is made as a permanent and perfect artificial cavitation.

Table 1 Parameters and amounts at three levels

Factors	Parameters	Level (+1)	Level (0)	Level (-1)
A	Cavitation number	1.5	1	0.5
B	Injection rate coefficient	0.45	0.32	0.18
C	Injection angle	60	30	0

5.1 Variance analysis

To determine the effect ratio of parameters on drag and lift, this test targets subjects so that dramatist models are made in the form of (3^3) 27. The second model 2FI and quadratic model are used to model the drag and lift coefficient. In models A, B, and C, cav, air, and tet correspond to the introducer cavitation number, injection rate coefficient, and injection angle, respectively. Variance analysis is confirmed in these models (Table 2). To examine the correctness of the relations among previous amounts in each drag and lift coefficient by deferent points, the relation between the internal and external extents for solving from the dramatist model improves the error percentages. Consistently, the internal extent is less than 4% and the external extent is less than 7% (Table 3).

The final relation to predict the lift and drag coefficients is obtained as follows:

$$\text{drag} = 0.36889 + 0.48694 \times \text{cav} - 0.45062 \times \text{air} - 0.0025 \times \text{tet} - 0.061728 \times \text{cav} \times \text{air} + \quad (3)$$

$$0.000138889 \times \text{cav} \times \text{tet} + 0.00761317 \times \text{air} \times \text{tet} \\ \text{lift} = +1.76346 - 3.77389 \times \text{cav} - 4.47051 \times \text{air} - 0.00124074 \times \text{tet} - 0.18519 \times \text{cav} \times \text{air} + \quad (4)$$

$$0.005 \times \text{cav} \times \text{tet} + 0.022634 \times \text{air} \times \text{tet} + 1.99111 \times \text{cav}^2 + 8.56577 \times \text{air}^2 - 0.000146914 \times \text{tet}^2$$

5.2 Drag coefficient graphs

As shown in Fig. 9, changes in the drag coefficient according to the injection angle and injection rate coefficient in the cavitation number are different. Each of the three cavitation numbers for a low injection rate coefficient (less than 28%) increases the injection angle because of the decrease in drag coefficient. However, an injection with a high injection rate coefficient (higher than 35%) increases the angle because of the increase in drag coefficient. In Fig. 10, the changes in drag coefficient are according to the injection angle and cavitation number in the injection rate coefficient. Fixed drag lines are similar to parallel lines. Grades of the drag coefficient lines are fixed in the injection rate coefficient at 18%. They gradually increase with the increase in injection

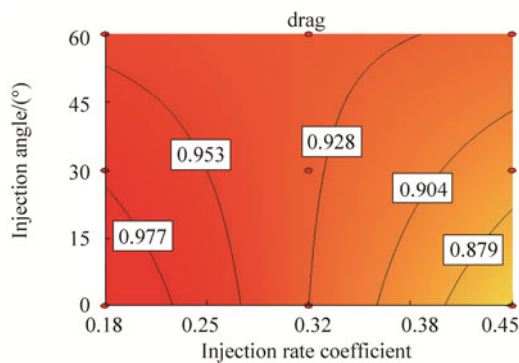
rate coefficient and increase to 45% when the injection rate coefficient is negative.

In these shapes, changes in the injection angle are not desirable (Fig. 11). In high injection rate coefficients, an

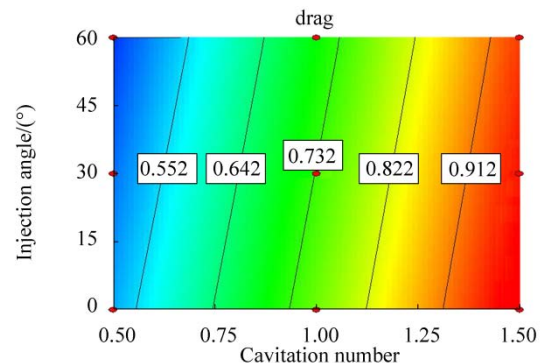
increase in injection angle causes an increase in an undesirable drag coefficient. Fixed drag lines are similar to parallel lines. Increasing the angle increases the grade of fixed drag coefficient lines.

Table 2 Variance analysis

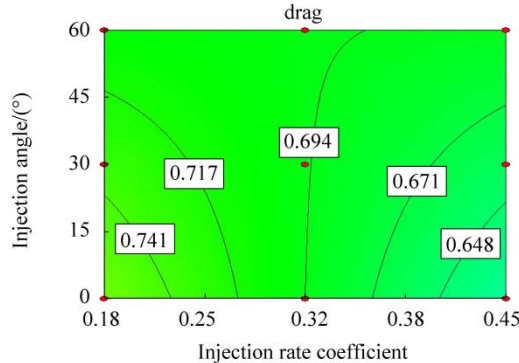
Source	Sum of squares	Degree of freedom	Mean square	F-value	p-value
Model	1.04	6	0.17	32.77	<0.000 1 significant
A-cav	1.00	1	1.00	189.40	<0.000 1 significant
B-air	0.026	1	0.026	5.00	0.003 68 significant
C-tet	2.222E-005	1	2.222E-005	4.204E-003	0.009 489 significant
AB	2.083E-004	1	2.083E-004	0.039	0.008 446 significant
AC	5.208E-005	1	5.208E-005	9.854E-003	0.009 219 significant
BC	0.011	1	0.011	2.16	0.001 574 significant



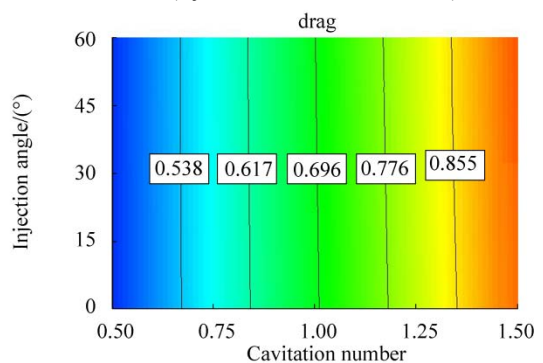
(a) Variations of drag coefficients according to the angle and injection rate coefficient (in cavitation number 1.5)



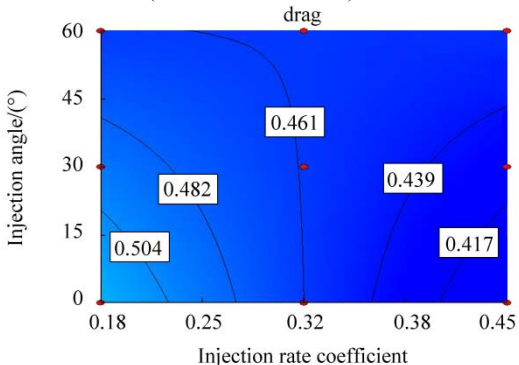
(a) Drag coefficient variations according to injection angle and cavitation number (injection rate coefficient of 0.18)



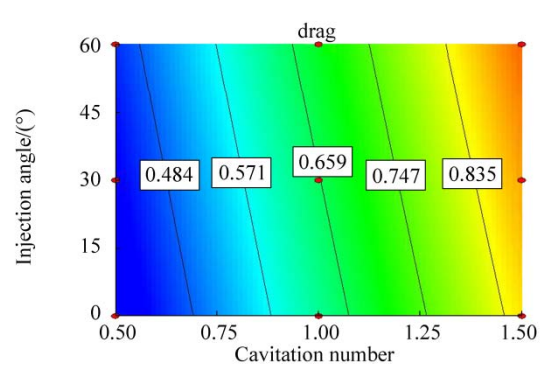
(b) Variations of drag coefficients according to the angle and injection rate coefficient (in cavitation number 1)



(b) Drag coefficient variations according to injection angle and cavitation number (injection rate coefficient of 0.32)



(c) Variations of drag coefficients according to the angle and injection rate coefficient (in cavitation number 0.5)



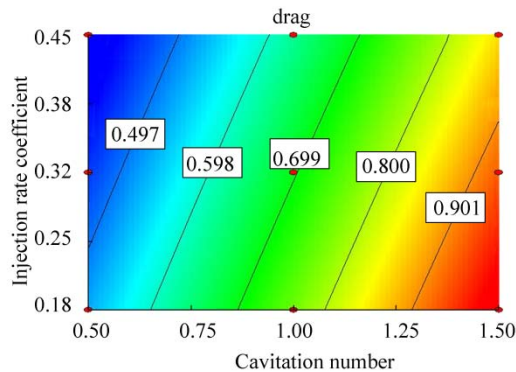
(c) Drag coefficient variations according to injection angle and cavitation number (injection rate coefficient of 0.45)

Fig. 9 Variations of drag coefficient according to the angle and injection rate coefficient in different cavitation numbers

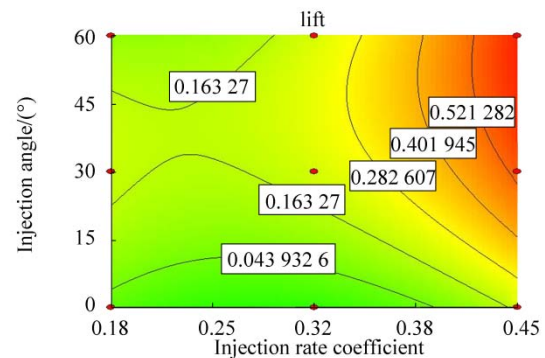
Fig. 10 Drag coefficient variations according to injection angle and cavitation number in different injection rate coefficients

Table 3 Percentage of error between the predicted values through the relationship and simulation value

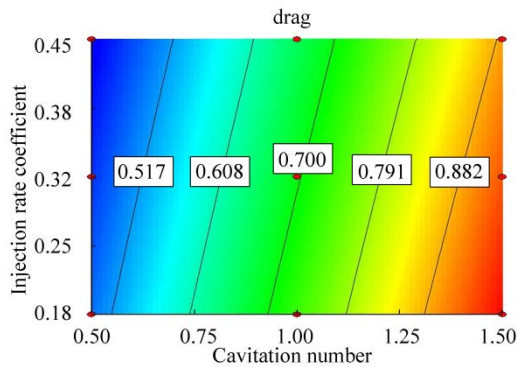
Variation range	Cavitation number	Injection rate	Injection angle	Maximization coefficient		Relation prediction coefficient		Error percentage	
				Lift	Drag	Lift	Drag	Lift	Drag
Within range	1.25	0.18	0	-0.4	0.87	-0.411 6	0.882 6	2.905 4	1.444 2
Within range	1.25	0.18	30	-0.28	0.85	-0.271 3	0.853 9	3.091 8	0.456 9
Within range	1.25	0.18	60	-0.4	0.86	-0.395 5	0.825 2	1.122 7	4.046 1
Within range	1.25	0.18	60	-0.27	0.82	-0.264 1	0.815 3	2.200 2	0.577 4
Within range	1	0.32	45	-0.45	0.72	-0.434 4	0.695 3	3.463 6	3.436 3
Within range	1.25	0.32	0	-0.45	0.82	-0.470 3	0.808 7	4.510 6	1.381 0
Without range	1	0.32	75	-0.6	0.725	-0.633 2	0.697 5	5.54	3.791 5
Without range	1.25	0.6	30	0.8	0.78	0.845 2	0.728 1	5.644	6.648 4



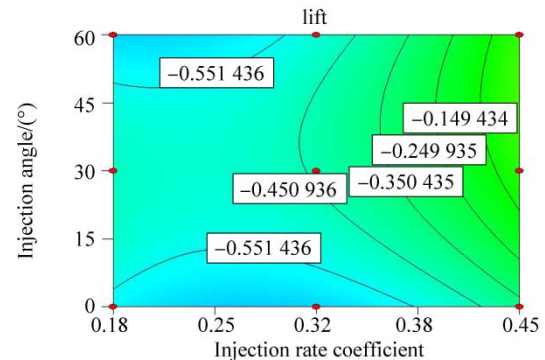
(a) Variations of drag coefficients according to injection rate coefficient and cavitation number (0° injection angle)



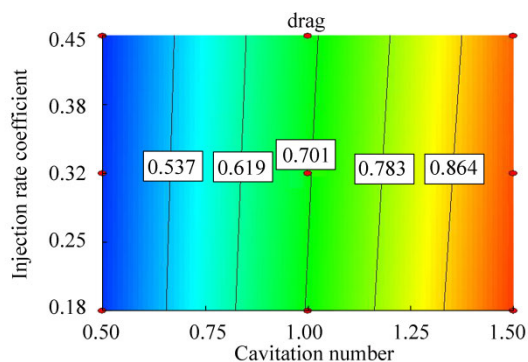
(a) Variations of lift coefficients according to angle and injection rate coefficient (in cavitation number 1.5)



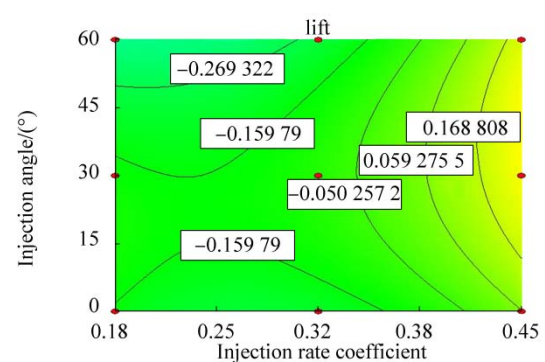
(b) Variations of drag coefficients according to injection rate coefficient and cavitation number (30° injection angle)



(b) variations of lift coefficients according to angle and injection rate coefficient (in cavitation number 1)



(c) Variations of drag coefficients according to injection rate coefficient and cavitation number (60° injection angle)



(c) variations of lift coefficient according to angle and injection rate coefficient (in cavitation number 0.5)

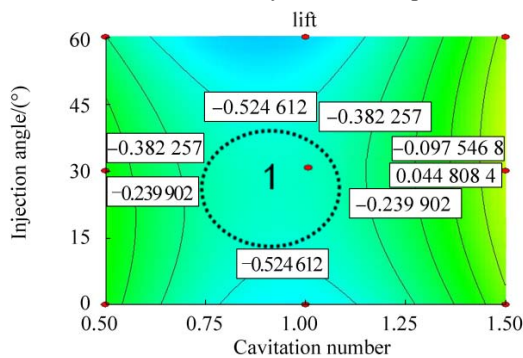
Fig. 11 Variations of drag coefficients according to injection rate coefficient and cavitation number in different degrees of injection angle**Fig. 12** Variations of lift coefficients according to angle and injection rate coefficient in different cavitation numbers

5.2 Lift coefficient graphs

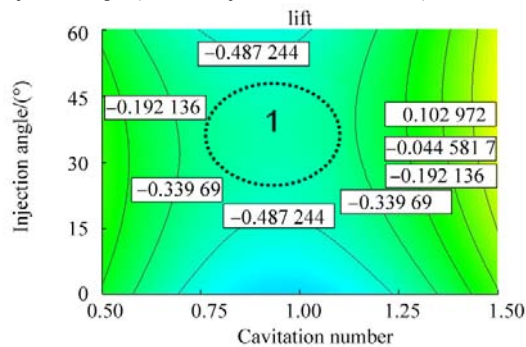
This section examines the graph changes in the lift coefficient in relation to the three parameters of injection rate, injection angle, and cavitation number. These changes depend on designers' ideas or on the application of a torpedo from the shot place to the up or down part or with no movement. The conclusion on these graphs is that they can reach the special target.

The lift coefficient increases in each of the three cavitation numbers by increasing the injection rate coefficient (Fig. 12). In a high injection rate coefficient, the change in lift coefficient is achieved by increasing the injection angle. Moreover, the lift coefficient increases with the increasing injection rate coefficient and injection angle. A high cavitation number means that the velocity of a low torpedo's change in

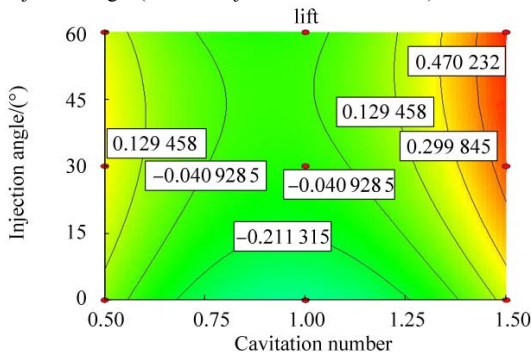
the extent of lift coefficient according to angle injection and injection rate coefficient has increased intensity, and a low cavitation number means decreased intensity. In this shape, the injection rate coefficient presented in the order of up to down increases more than the other results in this shape. In each cavitation number, the extent of change of the lift coefficient according to cavitation number and injection angle is small. For injection rate coefficients of 18% and 32%, and injection angle of 30° is observed, and the lift coefficient in the extent of change in cavitation number is fixed (Fig. 13). The extent of changes of the lift coefficient in Fig. 14 is in the order of up to down with a decreased injection angle. In this figure, the results indicate that the lift coefficient lines according to cavitation number and injection rate coefficient increase similar to concentric circles.



(a) Variations of lift coefficients according to cavitation number and injection angle (in 0.18% injection rate coefficient)

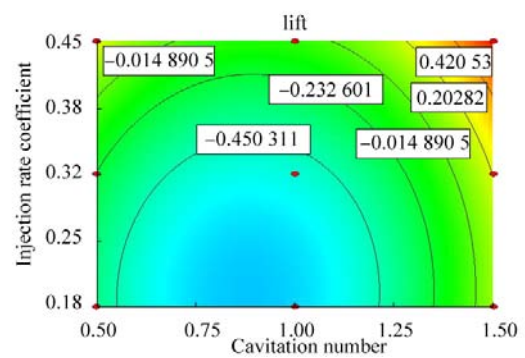


(b) Variations of lift coefficients according to cavitation number and injection angle (in 0.32% injection rate coefficient)

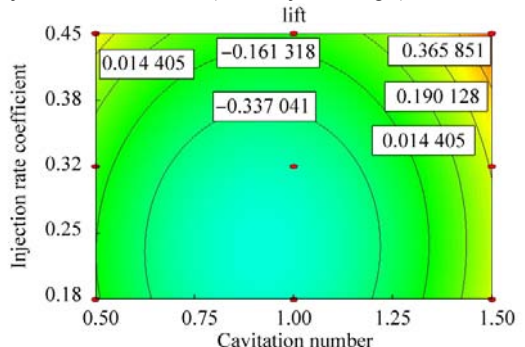


(c) Variations of lift coefficients according to cavitation number and injection angle (in 0.45% injection rate coefficient)

Fig. 13 Variations of lift coefficients according to cavitation number and injection angle in different degrees of injection rate coefficient



(a) Variations of lift coefficients based on cavitation number and injection rate coefficient (in 60° injection angle)



(b) Variations of lift coefficients based on cavitation number and injection rate coefficient (in 30° injection angle)

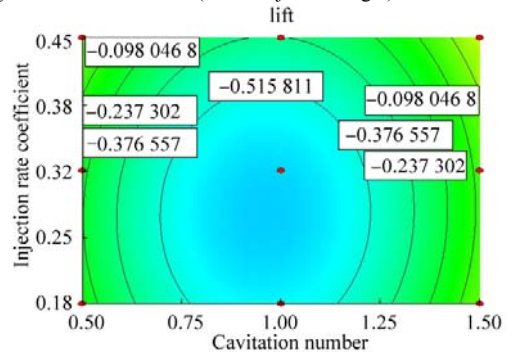


Fig. 14 Variations of lift coefficient based on cavitation number and injection rate coefficient in different degrees of injection angle

6 Conclusions

This study uses a test plan to determine the condition operation. Tests examining the operation of the three basic parameters of cavitation number, injection rate coefficient, and injection angle are performed at three levels. The end relation for drag (using the 2FI model) and lift coefficients (using the quadratic model) according to these parameters is attained. Then, the comparison between the calculation amount of these relations with the amount produced from the dramatist correctness relation (with errors less than 4% in the internal solution extent and less than 7% in the external solution extent) is discussed.

The mutual effects of cavitation number, injection rate coefficient, and injection angle on the lift and drag coefficients are in accordance with the relationships of different diagrams. These diagrams enable designers to change the angle and injection rate coefficient in different torpedo velocities (cavitation number) according to their purpose. As indicated in the diagrams, if the injection rate has an angle, it causes a remarkable improvement in decreasing the drag factor. However, in increasing the injection rate, the angle should be decreased to prevent the increase in drag coefficient.

Nomenclature

P_{∞}	Absolute pressure, N/m ²
Q_g	Gas volume flow, m ³ /s
q_g	Gas injection rate coefficient
V_{∞}	Reference velocity, m/s ²
σ_c	Artificial cavitation number
ρ	Water density, kg/m ³
D	Largest cavity diameter, m
D_n	Cavitator diameter, m
L	Cavity length, m
P_c	Pressure inside the bulb, N/m ²
P_v	Steam pressure, N/m ²
σ_v	Natural cavitation number

References

- Ahn BK, Lee TK, Kim HT, Lee CS, 2012. Experimental investigation of supercavitating flows. *International Journal of Naval Architecture and Ocean Engineering*, **4**(2), 123-131. DOI: 10.3744/JNAOE.2012.4.2.123
- Alishahi MM, 2010. The effect of the injected gas at super- gas injection (with the first leak). *The Tenth Conference of Iranian Aerospace Society*, 5.
- Baradaran Fard M, Nikseresht AH, 2012. Numerical simulation of unsteady 3D cavitating flows over axisymmetric cavitators. *Scientia Iranica*, **19**(5), 1258-1264. DOI: 10.1016/j.scient.2012.07.013
- Goel A, 2002. *Control strategies for supercavitating vehicles*. PhD thesis, University of Florida, Gainesville.
- Hashem Abadi SH, Dehnavi MA 2011. *CFD simulation of multiphase flows with Fluent software*. Andishe Sara, 208.
- Schauer TJ, 2003. *An experimental study of a ventilated supercavitating vehicle*. PhD thesis, University of Minnesota.
- Ma C, Jia D, Qian ZF, Feng DH, 2006. Study on cavitation flows of underwater vehicle. *Journal of Hydrodynamics*, **18**(3), 373-377. DOI: 10.1016/S1001-6058(06)60081-4
- Ji B, Luo XW, Peng XX, Zhang Y, Wu YL, Xu HY, 2010. Numerical investigation of the ventilated cavitating flow around an under-water vehicle based on a three-component cavitation model. *Journal of Hydrodynamics*, **22**(6), 753-759. DOI: 10.1016/S1001-6058(09)60113-X
- Roohi E, Zahir AP, Passandideh-Fard M, 2013. Numerical simulation of cavitation around a two-dimensional hydrofoil using VOF method and LES turbulence model. *Applied Mathematical Modelling*, **37**(9), 6469-6488. DOI: 10.1016/j.apm.2012.09.002
- Vlasenko YD, Savchenko GY, 2012. Study of the parameters of a ventilated supercavity closed on a cylindrical body. In: *Supercavitation*, Springer, 201-214. DOI: 10.1007/978-3-642-23656-3_11
- Wei YJ, Cao W, Wang C, Zhang JZ, Zou ZZ, 2009. Experimental research on character of ventilated supercavity. *New Trends in Fluid Mechanics Research*, Springer, Berlin, Heidelberg, 348-351. DOI: 10.1007/978-3-540-75995-9_108
- Yang WG, Zhang YW, Kan L, Deng F, 2009. Experimental investigation on the property of high-speed ventilated supercavitation. *New Trends in Fluid Mechanics Research*, Springer, Berlin, Heidelberg, 475-478. DOI: 10.1007/978-3-540-75995-9_152
- Zhang XW, Wei YJ, Zhang JZ, Wang C, Yu KP, 2007. Experimental research on the shape characters of natural and ventilated supercavitation. *Journal of Hydrodynamics*, **19**(5), 564-571. DOI: 10.1016/S1001-6058(07)60154-1
- Zou W, Yu KP, Wan XH, 2010. Research on the gas-leakage rate of unsteady ventilated supercavity. *Journal of Hydrodynamics* **22**(5), 778-783. DOI: 10.1016/S1001-6058(10)60030-3